



## Fan System Design and Performances Prediction Through Optimization Process

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# 1 – Industrial background

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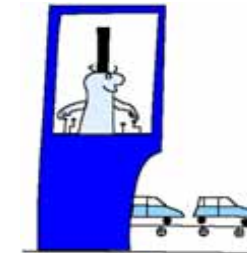
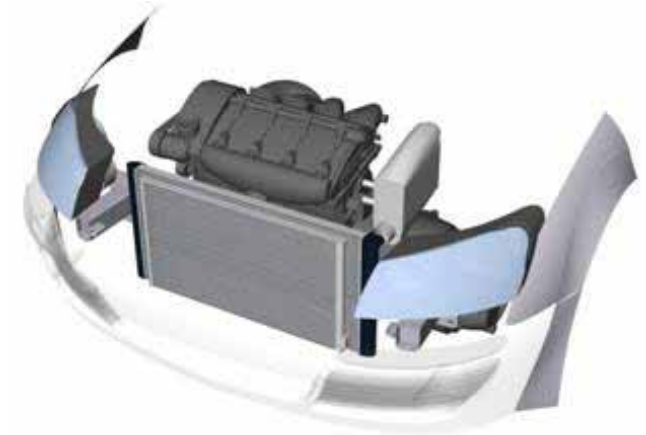
Components



Module



Front-end



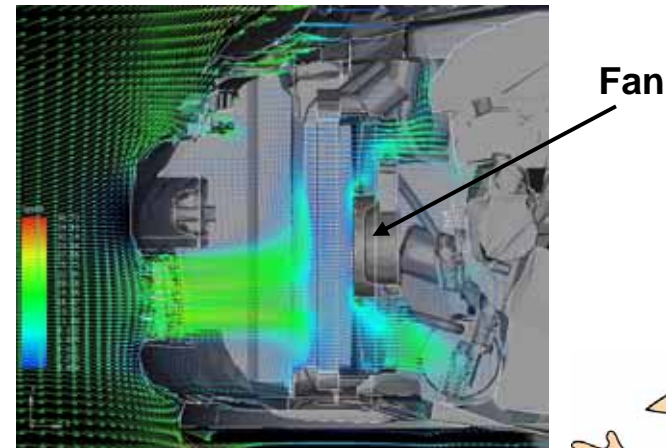
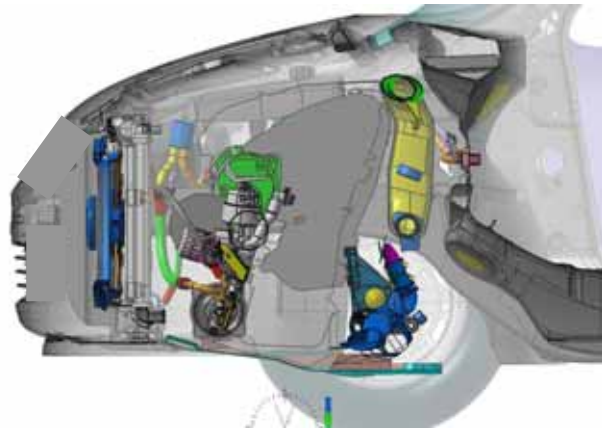
## ■ Fan system product lines and thermal components

- Components intended for mass production
- High pressure for reducing the cost of these products

# Technical constraints for fan system

## ■ More and more constraint of packaging in the underhood

- The downstream flow of the fan is affected by the aerodynamic blockage created by mechanical components (motor, gear-box, ...)
- The fan is no longer working as a standard axial turbomachinery



## ■ Need to improve development methods

- Standard theory for axial machine not fully valid for such flow
- For energy efficiency, the fan must provide optimum performances

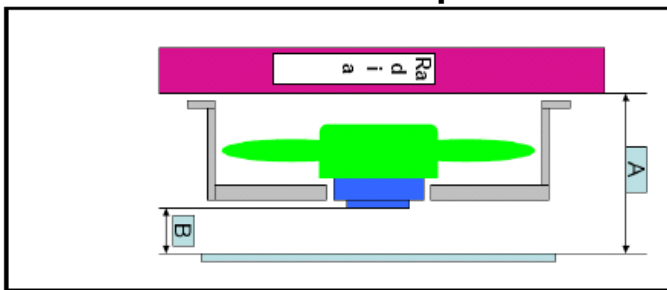
## 2 – Objectives

# New type of specifications for fan system

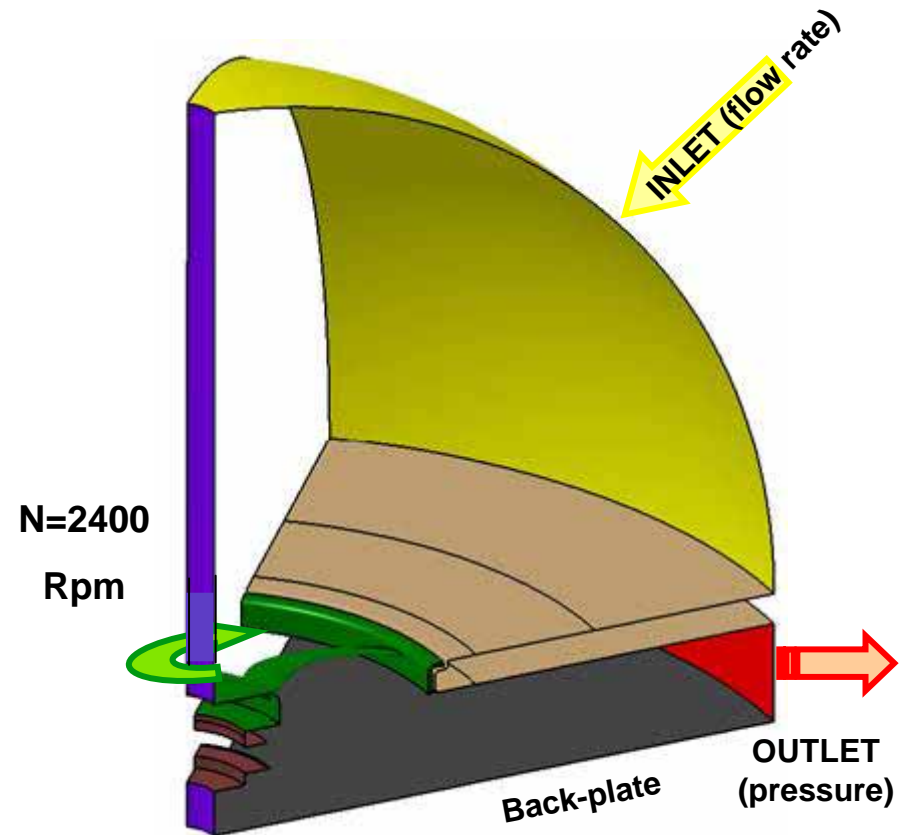
## ■ Fluid simulation intended to improve fan systems

- Simulation can take into account complex environments
- Realistic configuration compared to real usage of the fan in the under hood

Black-plate configuration:  
120 mm Hx to plate



Specifications



Only one blade passage meshed (1/7)

Domaine of simulation

# Build a methodology to conduct complex development

## ■ Combine optimization method with aerodynamic fan simulation

- Build the methodology on the base of the know-how of experts
- Link the CFD tool CFD++ (Metacomptech) with Hyperworks platform (Altair)
- Provide solutions for best performances with a high level of confidence

## ■ Experiment and test the method to be implemented with high power computing

- Demonstrate the benefit of big computing capacities (Expamtion Project)
- Save time and resources to go further in our research (acoustic for instance with CAA++...)



## 3 – Methods

# The different steps

## ■ Sensitivity analysis

- Several parameters are chosen on the base of the know-how of experts
- A first DOE is conducted with coarse meshes
- Results analysis help ranking parameters and determine cross-correlation between them

## ■ Optimization conducted with a reduce set of parameters

- Most influent parameters are selected to drive the optimization with fine meshes
- The objective is to minimise the torque
- Two constraints are established:
  - Pressure rise  $> 210$  Pa at nominal point (QN)
  - Pressure rise  $> 0$  at high flow rate (QH)

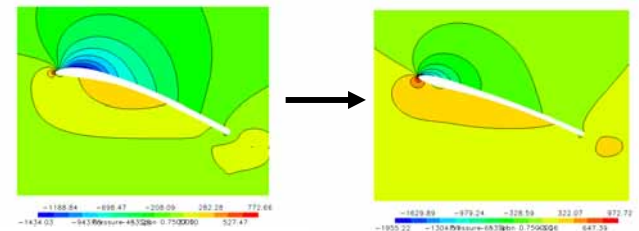
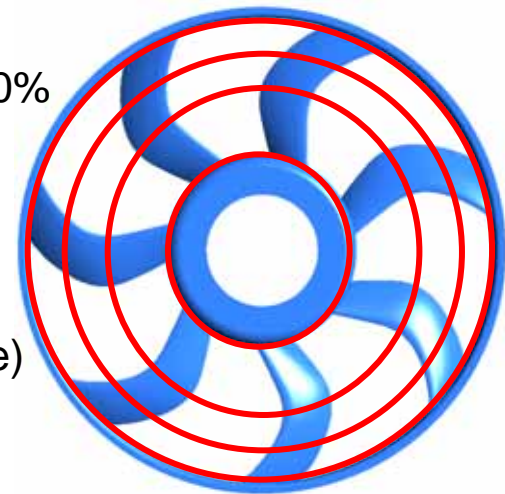
## ■ Optimization conducted with a full set of parameters (to be continued)

- Demonstrate the ability of using highly parallelised computing to reduce development time
- Evaluate the cost and benefit of running the full set of parameter compared to the reduce one

# Parameters

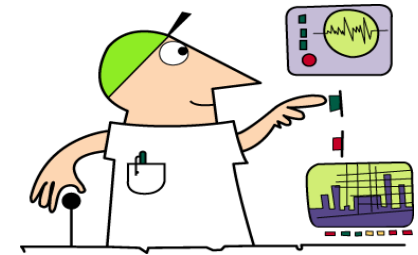
## ■ Chosen Parameters to describe a complex blade surface

- 4 selected radius to obtain the effect of the twist: bottom, mid span, 80% of the span, top
- For each radius, chord can be move from 80% to 120%
- For each radius, camber can be modified from 100% to 20%
- For each radius, stagger can be changed by +/- 4°
- A total of 12 independent parameters
- 2 different flow rates (the nominal one and a high flow rate)
- Initial geometry based on a existing one
- Blade thickness distribution kept constant
- Sweep variation not studied

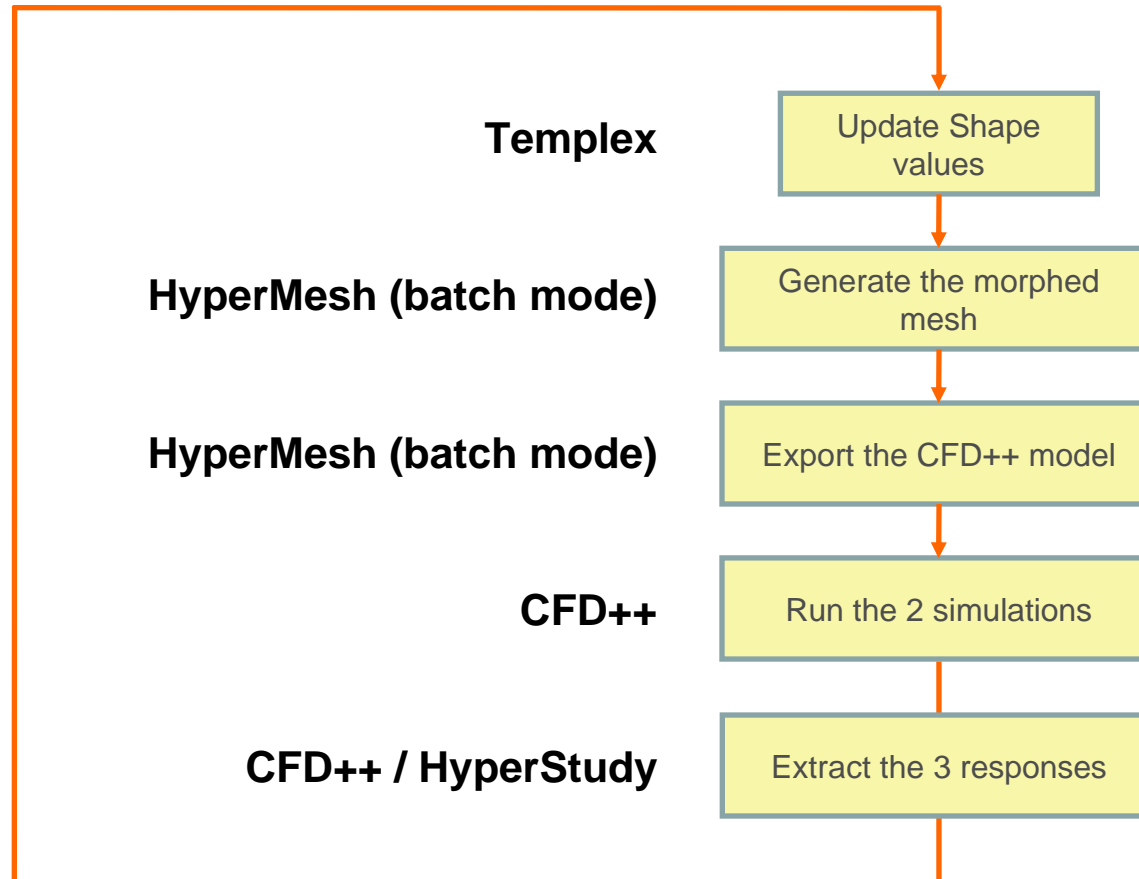


**Variations can be considered as small  
→ Local optimization in the field of solutions**

# Process



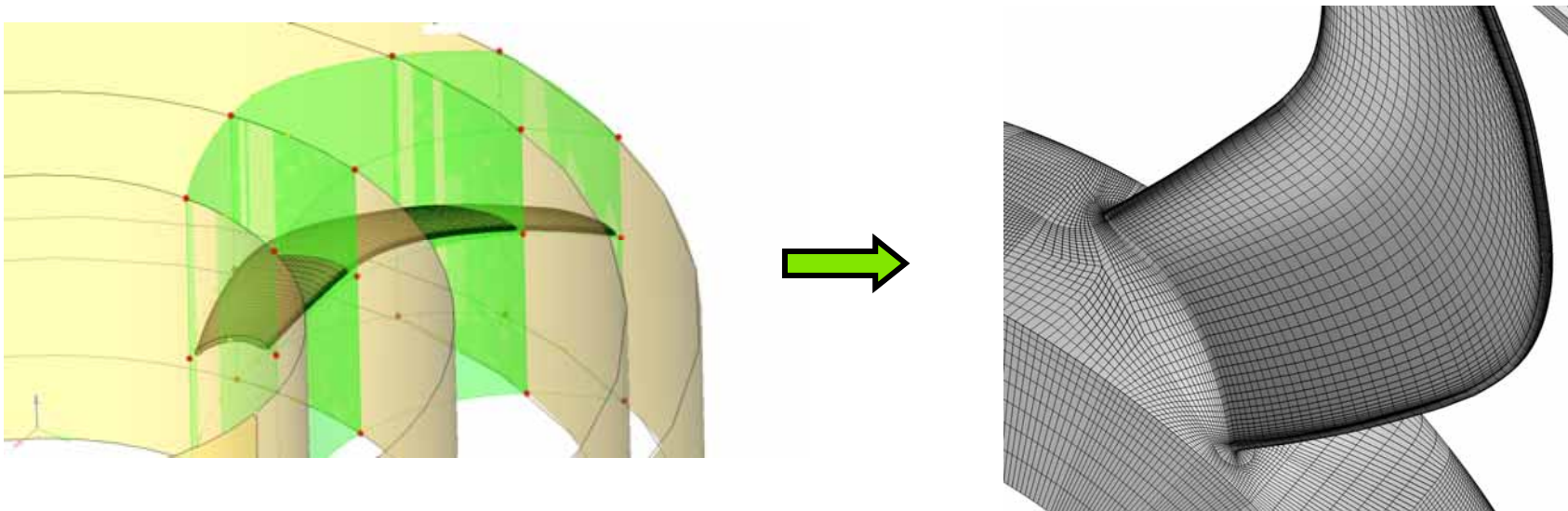
## HyperStudy Optimization Algorithm



# Mesh morphing

## ■ Process to deform the mesh (full hexa meshing)

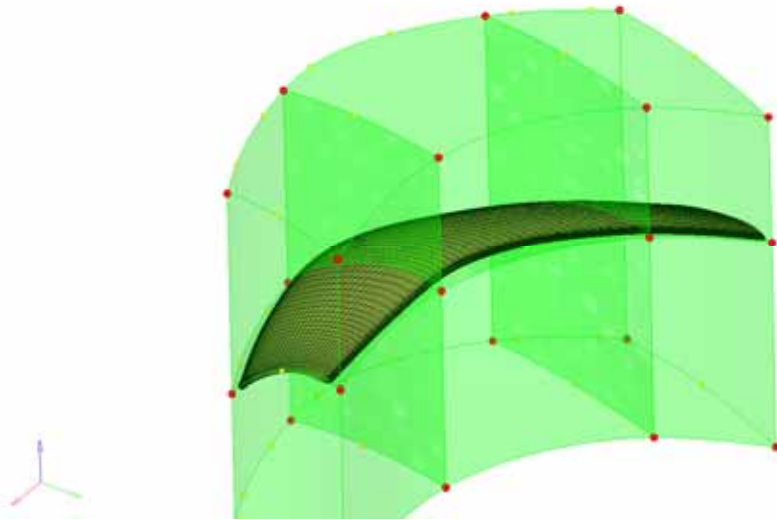
- The blade surface mesh is deformed by the help of handles
- The mesh around the blade aimed to catch the boundary flow is morphed according to the new shape (O-grid topology)
- The external domain is reconstructed to kept constant dimensions although the package of the fan has changed
- Automated re-meshing between the O-grid and the external limits of the domain



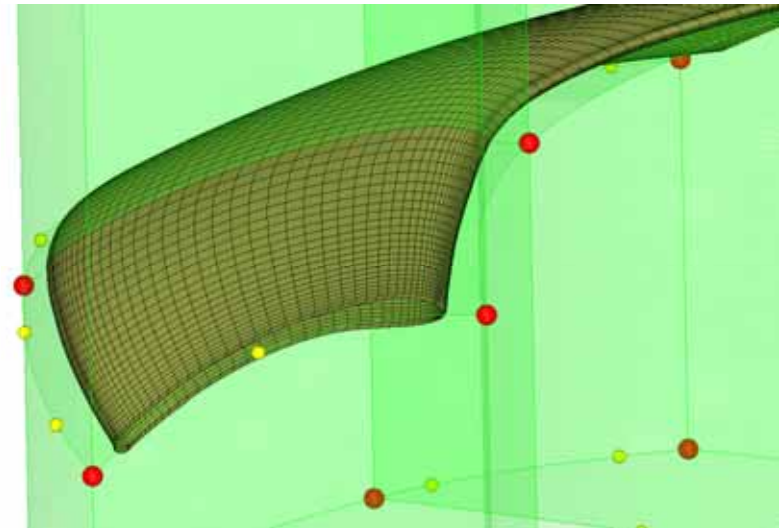
# Mesh morphing

## ■ Handles control the morphing

- Handles placed according to the description of a aerodynamic profile: leading edge, trailing edge, mid chord
- Handles are located at for different radius (bottom, mid-span,  $\frac{3}{4}$  span and top)



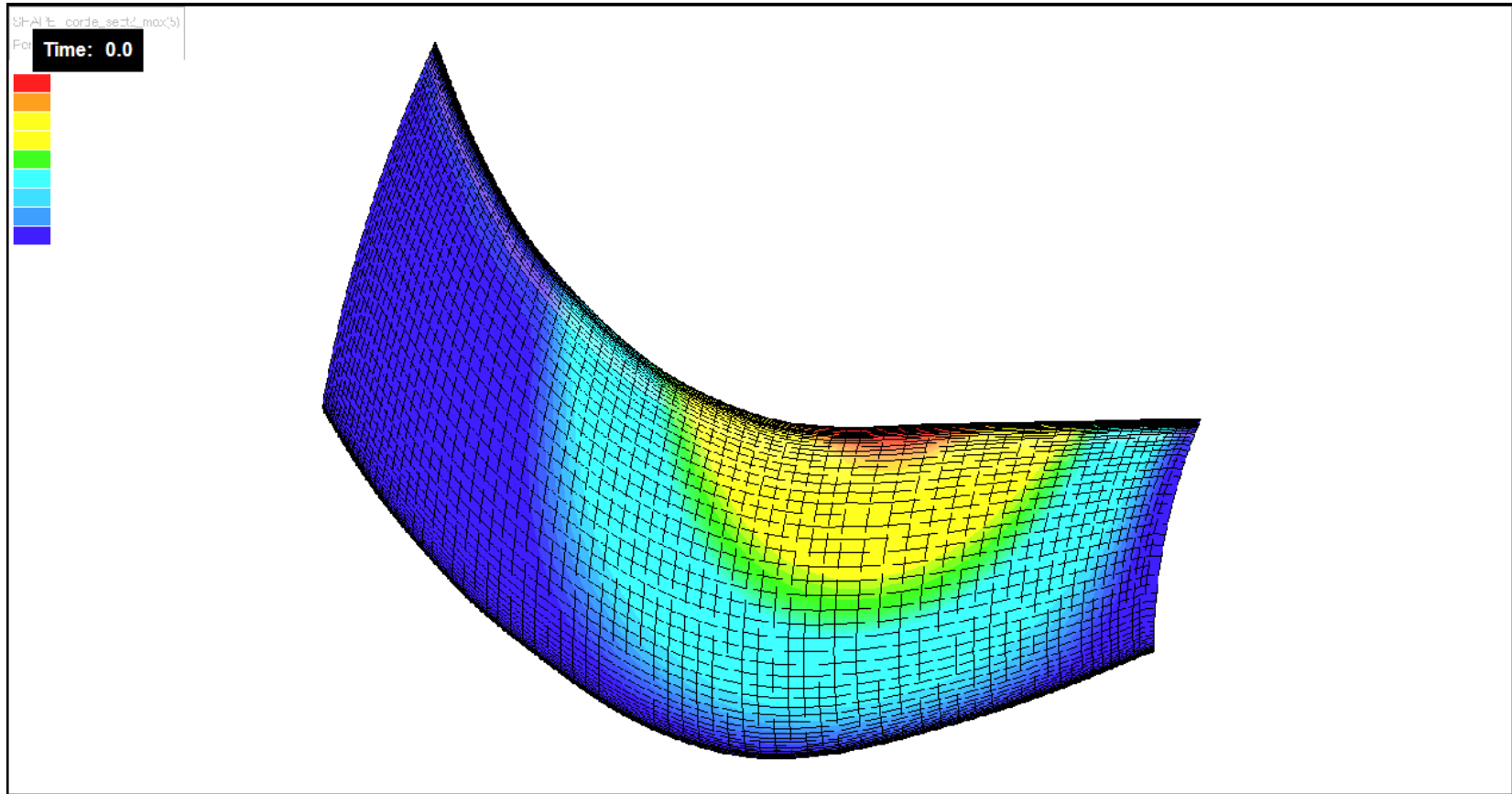
General view of the blade with 4 chosen radius



Locations of handles:  
Red = master (can be move independently)  
Yellow = slave

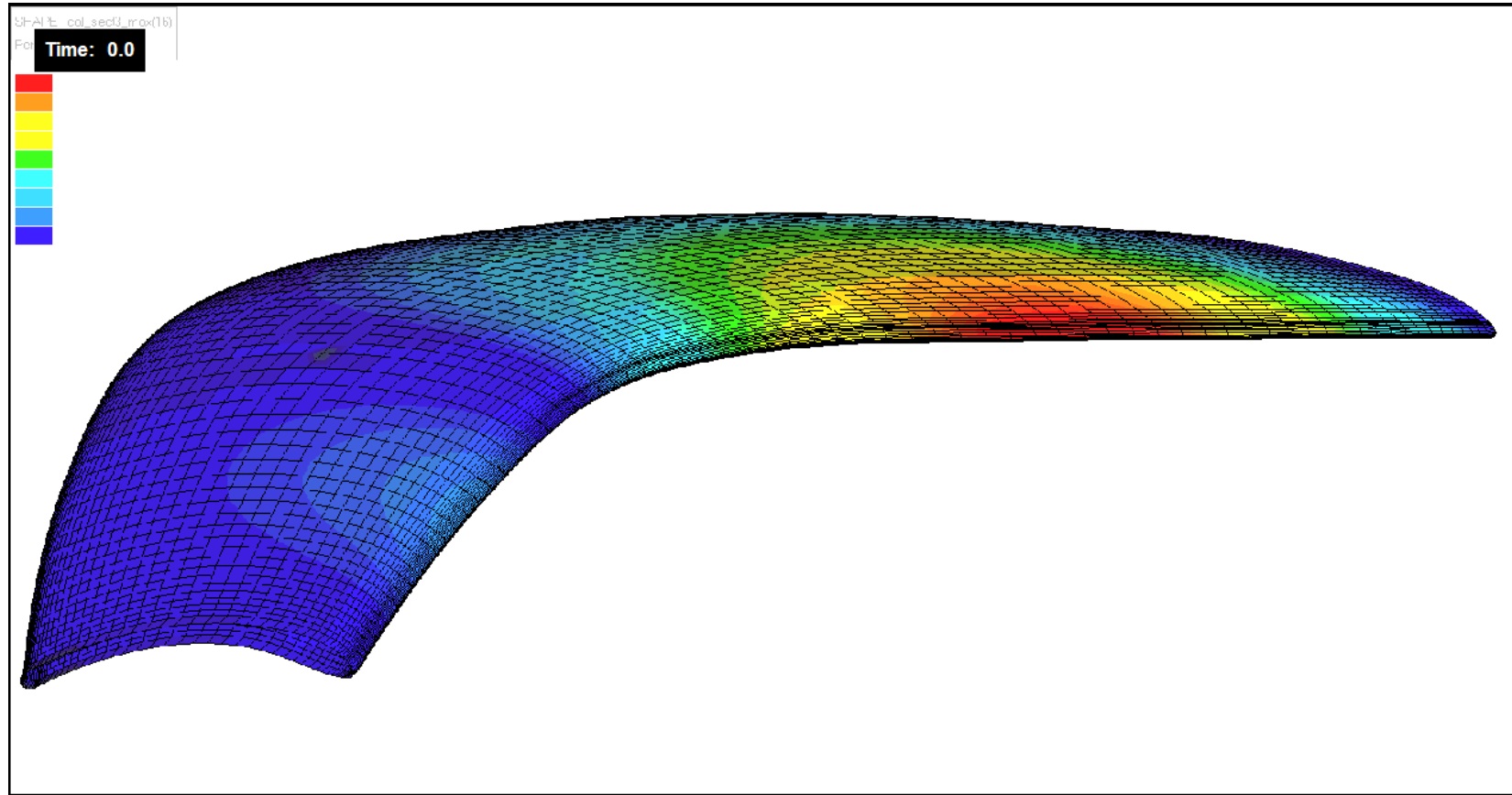
# Mesh morphing

## ■ Chord variation



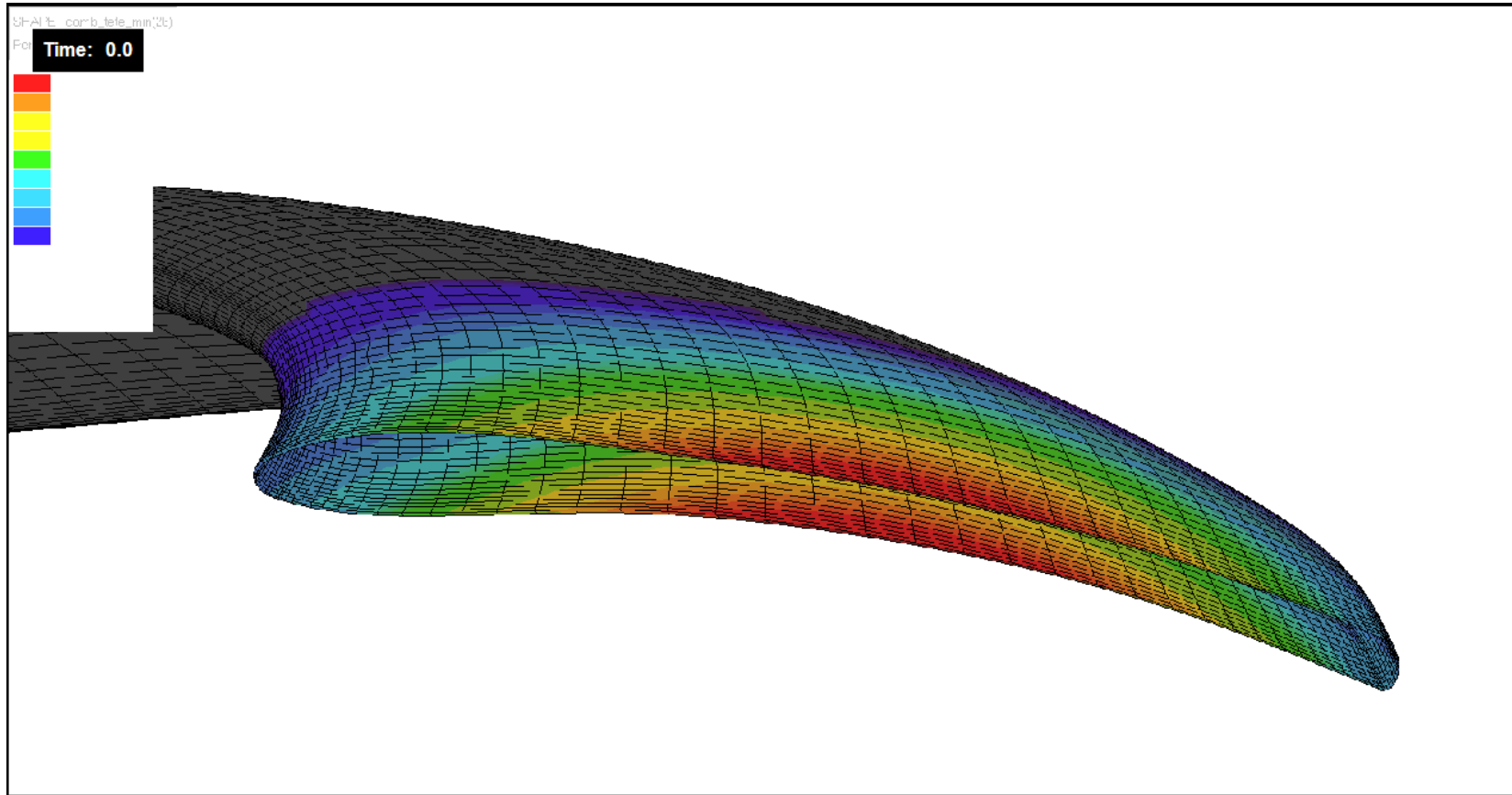
# Mesh morphing

## ■ Stagger angle variation



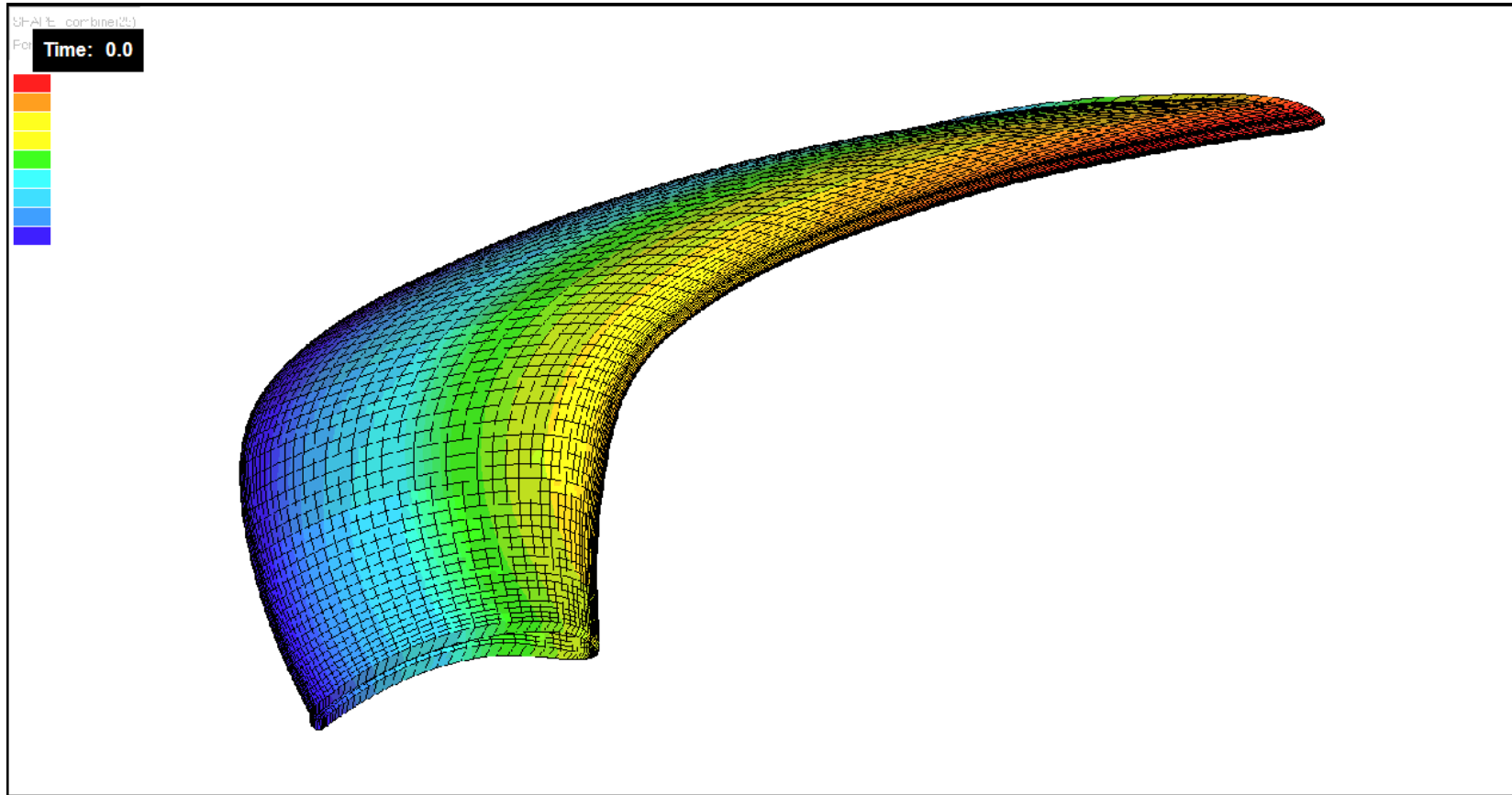
# Mesh morphing

## ■ Camber variation



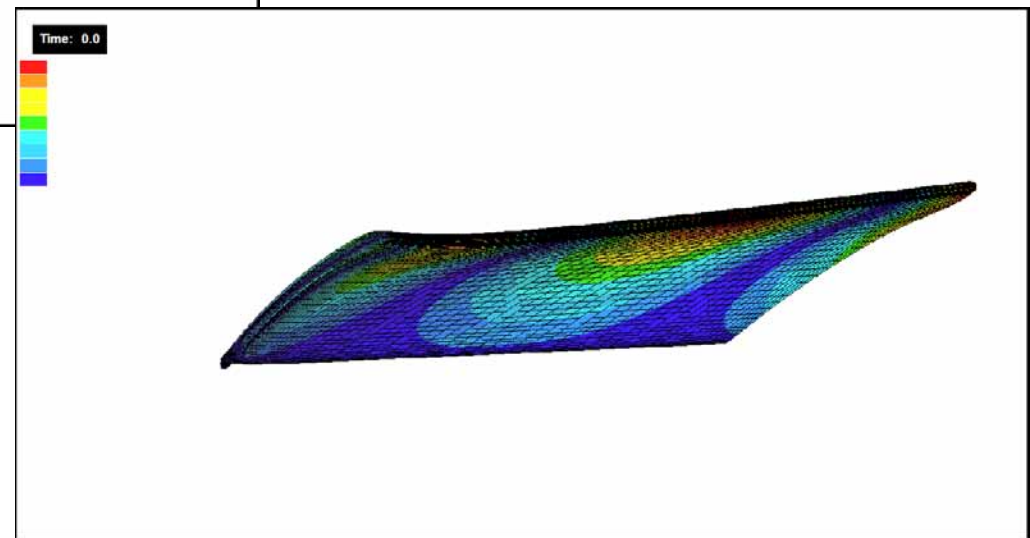
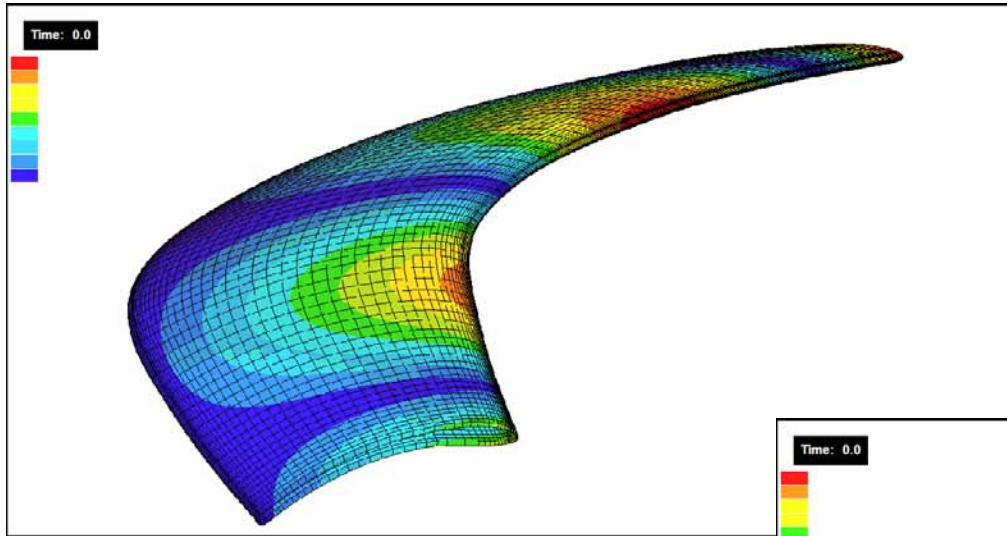
# Mesh morphing

## ■ Maximum envelope



# Mesh morphing

## ■ Some variations

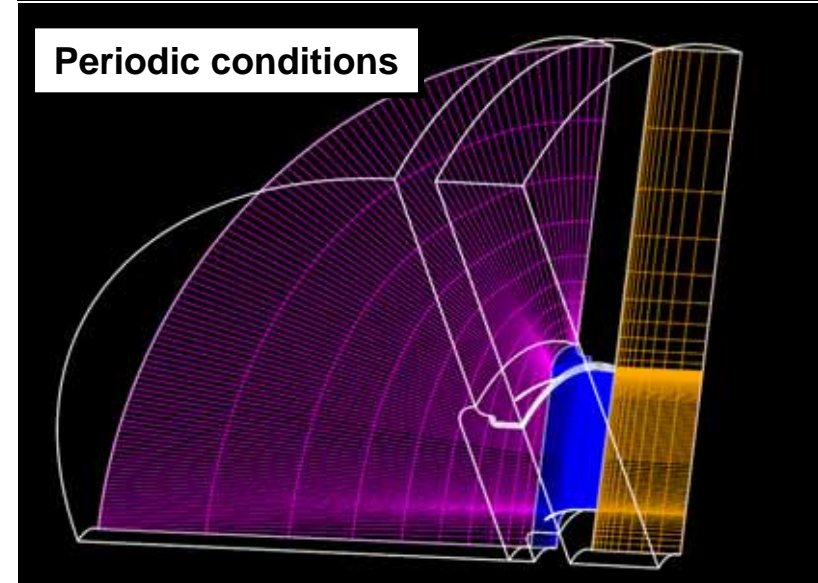
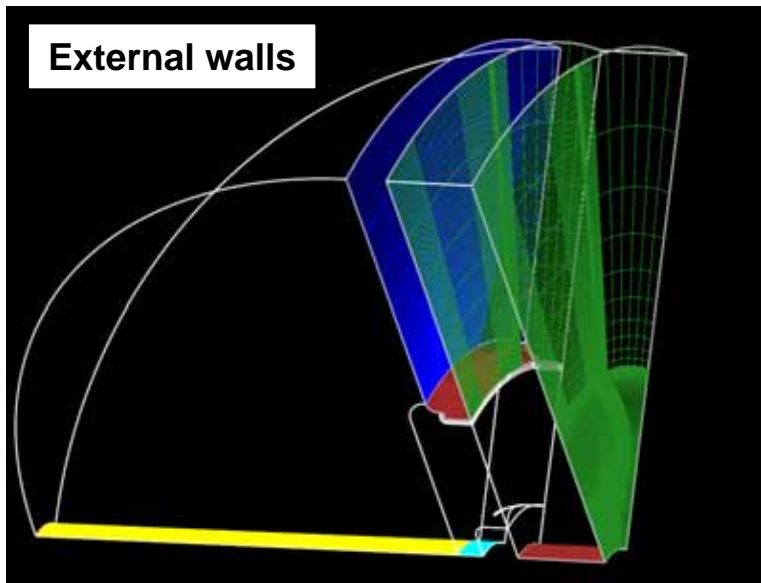
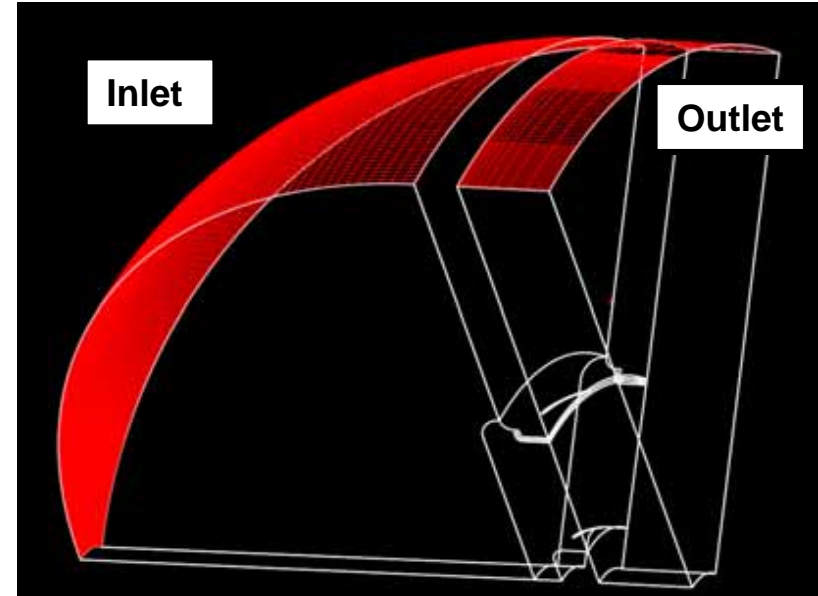
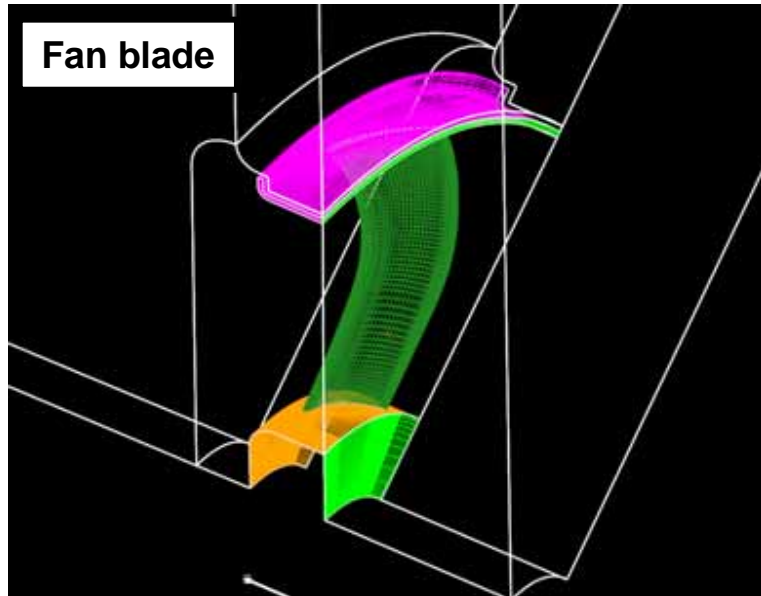


Objective: give a maximum of freedom to the shape

# Fan simulation (CFD++)

Mesh size: 616 500 nodes / 585 000 cells

(Inlet: 92 500 cells / Fan: 398 000 cells / Outlet: 94 000 cells)



## 4 – DOE and sensitivities

# First set of simulations

- NOLH designs (*Nearly Orthogonal Latin Hypercube*)
- 65 designs
- Each parameter has only one time the same value
- Theoretically, the distances between each design are maximized

Rayon	85,3	131,88	186,21	209,5	85,3	131,88	186,21	209,5	85,3	131,88	186,21	209,5
copie	0,9	0,9	0,9	0,9	-4	-4	-4	-4	0	0	0	0
	1,1	1,1	1,1	1,1	4	4	4	4	0,8	0,8	0,8	0,8
	2	2	2	2	2	2	2	2	2	2	2	2
	corde 1	corde 7	corde 14	corde 17	calage 1	calage 7	calage 14	calage 17	camber 1	camber 7	camber 14	camber 17
1	1,04	0,91	0,97	0,97	-3	2,13	2,38	-0,13	0,78	0,58	0,44	0,75
2	1,09	1,04	0,92	0,98	-1,25	-2	0,38	2	0,58	0,74	0,61	0,39
3	1,08	0,97	1,09	0,94	-1,63	2,88	-2,75	-0,38	0,33	0,46	0,64	0,7
4	1,03	1,08	1,04	0,99	-3,5	-0,63	-1,88	-1,88	0,1	0,79	0,73	0,5
5	1,08	0,99	0,94	0,9	-3,25	-2,75	-2	0,88	0,49	0,39	0,05	0,68
6	1,01	1,08	0,95	1	-2,75	1,25	-2,88	2,88	0,71	0,09	0,33	0,63
7	1,05	0,94	1,01	0,91	-2	-1,38	3,13	-0,63	0,13	0,35	0,28	0,73
8	1,06	1,05	1,08	0,96	-1	3,38	1,63	-3,88	0,2	0,25	0,01	0,51
9	1,04	0,91	0,9	1,06	-0,75	0,88	-0,88	-3,25	0,64	0,24	0,68	0,31
10	1,09	1,04	1	1,05	-4	-1,75	3,38	-1	0,54	0	0,54	0,03
11	1	0,9	1,09	1,01	-2,38	-0,5	-1,5	1,5	0,01	0,19	0,58	0,15
12	1,1	1	1,04	1,08	-2,63	-3,5	-1	2,5	0,25	0,3	0,43	0,2
13	1,01	0,94	0,97	1,03	-0,88	-0,13	-2,5	-1,75	0,76	0,76	0,11	0,04
14	1,06	1,01	0,98	1,08	-1,88	3,75	-3,63	-4	0,41	0,43	0,2	0,24
15	1,02	0,97	1,06	1,09	-0,5	-2,25	-0,25	3,5	0,11	0,66	0	0,35
16	1,03	1,02	1,01	1,04	-2,88	3,63	4	0,5	0,36	0,65	0,29	0,09
17	1,07	0,99	0,98	0,93	1,38	4	2,25	3,75	0,46	0,48	0,25	0,23
18	1,01	1,07	0,95	0,97	0,63	-0,38	2,63	2,38	0,29	0,59	0,09	0

# Second set of simulations

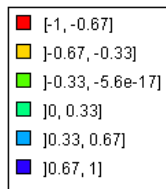
- Extreme values + Hammersley run
- 116 designs
- Fitted for design of experiment

Complement extrême -->	66	0,9000	0,9000	0,9000	0,9000	4,0000	-4,0000	-4,0000	4,0000	0,0000
Complement extrême -->	67	1,1000	0,9000	0,9000	0,9000	-4,0000	-4,0000	4,0000	4,0000	0,8000
Complement extrême -->	68	0,9000	1,1000	0,9000	0,9000	-4,0000	4,0000	-4,0000	4,0000	0,8000
Complement extrême -->	69	1,1000	1,1000	0,9000	0,9000	4,0000	4,0000	4,0000	4,0000	0,0000
Complement extrême -->	70	0,9000	0,9000	1,1000	0,9000	-4,0000	4,0000	4,0000	-4,0000	0,0000
Complement extrême -->	71	1,1000	0,9000	1,1000	0,9000	4,0000	4,0000	-4,0000	-4,0000	0,8000
Complement extrême -->	72	0,9000	1,1000	1,1000	0,9000	4,0000	-4,0000	4,0000	-4,0000	0,8000
Complement extrême -->	73	1,1000	1,1000	1,1000	0,9000	-4,0000	-4,0000	-4,0000	-4,0000	0,0000
Complement extrême -->	74	0,9000	0,9000	0,9000	1,1000	-4,0000	4,0000	4,0000	-4,0000	0,8000
Complement extrême -->	75	1,1000	0,9000	0,9000	1,1000	4,0000	4,0000	-4,0000	-4,0000	0,0000
Complement extrême -->	76	0,9000	1,1000	0,9000	1,1000	4,0000	-4,0000	4,0000	-4,0000	0,0000
Complement extrême -->	77	1,1000	1,1000	0,9000	1,1000	-4,0000	-4,0000	-4,0000	-4,0000	0,8000
Complement extrême -->	78	0,9000	0,9000	1,1000	1,1000	4,0000	-4,0000	-4,0000	4,0000	0,8000
Complement extrême -->	79	1,1000	0,9000	1,1000	1,1000	-4,0000	-4,0000	4,0000	4,0000	0,0000
Complement extrême -->	80	0,9000	1,1000	1,1000	1,1000	-4,0000	4,0000	-4,0000	4,0000	0,0000
Complement extrême -->	81	1,1000	1,1000	1,1000	1,1000	4,0000	4,0000	4,0000	4,0000	0,8000
un tirage dit de Hammersley	1	0,9011	1,0728	0,9508	1,0295	2,4134	0,5059	1,1689	1,1459	0,1643
un tirage dit de Hammersley	2	0,9022	1,0148	0,9176	0,9013	-1,1999	0,6841	0,5373	-0,1887	0,1163
un tirage dit de Hammersley	3	0,9033	0,9255	1,0619	1,0481	1,4244	3,9262	0,4149	-1,5214	0,2217
un tirage dit de Hammersley	4	0,9044	1,0071	0,9328	0,9776	-1,4591	-1,2378	0,4956	-1,0296	0,6303
un tirage dit de Hammersley	5	0,9055	1,0957	0,9670	0,9909	-2,8725	-2,9943	2,0093	-1,9189	0,1705
un tirage dit de Hammersley	6	0,9066	0,9933	0,9808	1,0250	-3,8134	1,3546	3,9562	-2,8254	0,0362
un tirage dit de Hammersley	7	0,9077	1,0338	0,9027	0,9417	-0,6654	-2,3189	-1,0741	3,2272	0,2176

# Result Analysis

## Correlation

- 65 designs (NOLH) + 116 designs (Hammersley)
- Low correlations between parameters: all are useful (green area)
- High correlation between objectives: non independent (blue area)

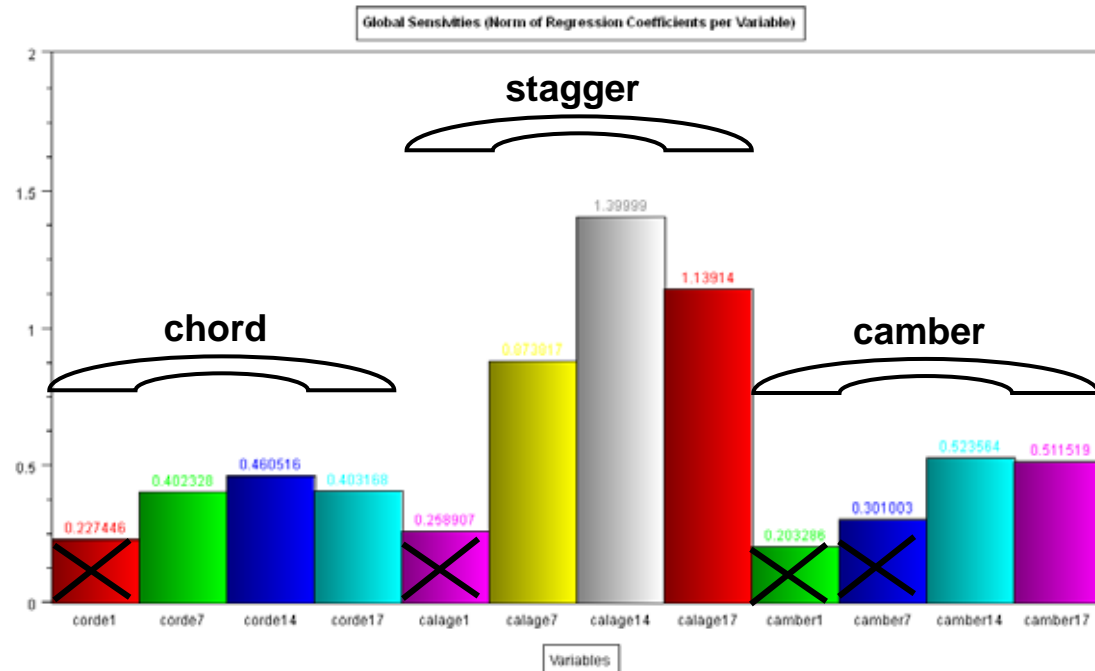


	Dp2300	T2300	Eff2300	Dp4000	T4000	Eff4000	corde1	corde7	corde14	corde17	calage1	calage7	calage14	calage17	camber1	camber7	camber14	camber17
Dp2300	1	0.96	0.95	0.82	0.88	0.8	0.15	0.15	0.12	0.22	0.091	0.39	0.6	0.52	0.18	0.079	0.019	-0.33
T2300	0.96	1	0.68	0.87	0.95	0.96	0.094	0.079	0.082	0.19	0.077	0.39	0.72	0.48	0.09	0.055	-0.088	-0.27
Eff2300	0.95	0.68	1	0.55	0.57	0.53	0.2	0.21	0.16	0.22	0.092	0.31	0.28	0.47	0.29	0.093	0.2	-0.36
Dp4000	0.82	0.87	0.55	1	0.9	0.98	0.025	0.18	0.094	0.15	0.005	0.45	0.6	0.52	0.1	-0.14	-0.24	0.06
T4000	0.88	0.95	0.57	0.9	1	0.9	0.1	0.092	0.066	0.12	0.094	0.34	0.72	0.45	0.079	-0.045	-0.3	-0.27
Eff4000	0.8	0.96	0.53	0.98	0.9	1	-0.0096	0.12	0.11	0.11	0.027	0.42	0.61	0.5	0.069	-0.14	-0.26	0.02
corde1	0.15	0.094	0.2	0.025	0.1	-0.0096	1	-0.018	-0.0075	0.0061	-0.036	0.014	0.056	0.019	0.11	0.017	0.022	0.017
corde7	0.15	0.079	0.21	0.18	0.092	0.12	-0.018	1	-0.12	0.026	-0.028	0.029	-0.041	0.023	-0.014	0.039	-0.062	-0.018
corde14	0.12	0.082	0.16	0.094	0.066	0.11	-0.0075	-0.12	1	0.029	-0.012	-0.13	0.0078	-0.044	0.075	-0.053	0.025	0.055
corde17	0.22	0.19	0.22	0.15	0.12	0.11	0.0061	0.026	0.029	1	-0.046	-0.038	0.1	-0.072	-0.011	-0.12	-0.0024	0.023
calage1	0.091	0.077	0.092	0.005	0.094	0.027	-0.036	-0.028	-0.012	-0.046	1	-0.0028	-0.027	-0.052	0.12	-0.1	-0.11	-0.03
calage7	0.39	0.39	0.31	0.45	0.34	0.42	0.014	0.029	-0.13	-0.038	-0.0028	1	0.0072	0.091	0.077	-0.057	0.041	-0.058
calage14	0.6	0.72	0.28	0.6	0.72	0.61	0.036	-0.041	0.0078	0.1	-0.027	0.0072	1	0.036	0.022	0.12	-0.036	-0.093
calage17	0.52	0.48	0.47	0.52	0.45	0.5	0.019	0.023	-0.044	-0.072	-0.052	0.091	0.036	1	0.034	0.034	0.063	-0.016
camber1	0.18	0.09	0.29	0.1	0.079	0.069	0.11	-0.014	0.075	-0.011	0.12	0.077	0.022	0.034	1	0.024	-0.042	-0.07
camber7	0.079	0.055	0.093	-0.14	-0.045	-0.14	0.017	0.039	-0.053	-0.12	-0.1	-0.057	0.12	0.034	0.024	1	0.0058	-0.1
camber14	0.019	-0.088	0.2	-0.24	-0.3	-0.26	0.022	-0.062	0.025	-0.0024	-0.11	0.041	-0.036	0.063	-0.042	0.0058	1	-0.069
camber17	-0.33	-0.27	-0.36	0.06	-0.27	0.02	0.017	-0.018	0.055	0.023	-0.03	-0.058	-0.093	-0.016	-0.07	-0.1	-0.069	1

# Result Analysis

## ■ Global sensitivities

- All parameters count, with different loads
- Stagger is the main parameter
- Chord 14 (80% span) is the most important location



## ■ Reduce set of parameters (8)

- Parameters at bottom can be rejected at first (chord, stagger, camber)
- Camber at mid span is also rejected

## 4 – Optimization

# Optimisation with Adaptive Surface Response Method (ARSM)

→ Parameters at bottom and camber at mid span are disabled

Design variables

	On	Label	Variable Name	Model Parameter	Variable Type
▶	<input type="checkbox"/>	dv1	m_1_corde_pied	m_1.corde_pied	Design
	<input type="checkbox"/>	dv2	m_1_cal_pied	m_1.cal_pied	Design
	<input type="checkbox"/>	dv3	m_1_camb_pied	m_1.camb_pied	Design
	<input checked="" type="checkbox"/>	dv4	m_1_corde_sect2	m_1.corde_sect2	Design
	<input checked="" type="checkbox"/>	dv5	m_1_cal_sect2	m_1.cal_sect2	Design
	<input type="checkbox"/>	dv6	m_1_camb_sect2	m_1.camb_sect2	Design
	<input checked="" type="checkbox"/>	dv7	m_1_corde_sect3	m_1.corde_sect3	Design
	<input checked="" type="checkbox"/>	dv8	m_1_cal_sect3	m_1.cal_sect3	Design
	<input checked="" type="checkbox"/>	dv9	m_1_camb_sect3	m_1.camb_sect3	Design
	<input checked="" type="checkbox"/>	dv10	m_1_corde_tete	m_1.corde_tete	Design
	<input checked="" type="checkbox"/>	dv11	m_1_cal_tete	m_1.cal_tete	Design
	<input checked="" type="checkbox"/>	dv12	m_1_camb_tete	m_1.camb_tete	Design

→ Minimize the torque (in absolute value)

Define objective

Goal: Maximize

	On	Label	Varname	Apply On	Evaluate from
▶	<input checked="" type="radio"/>	Objective_1	obj_1	torque_rpm230	SOLVER

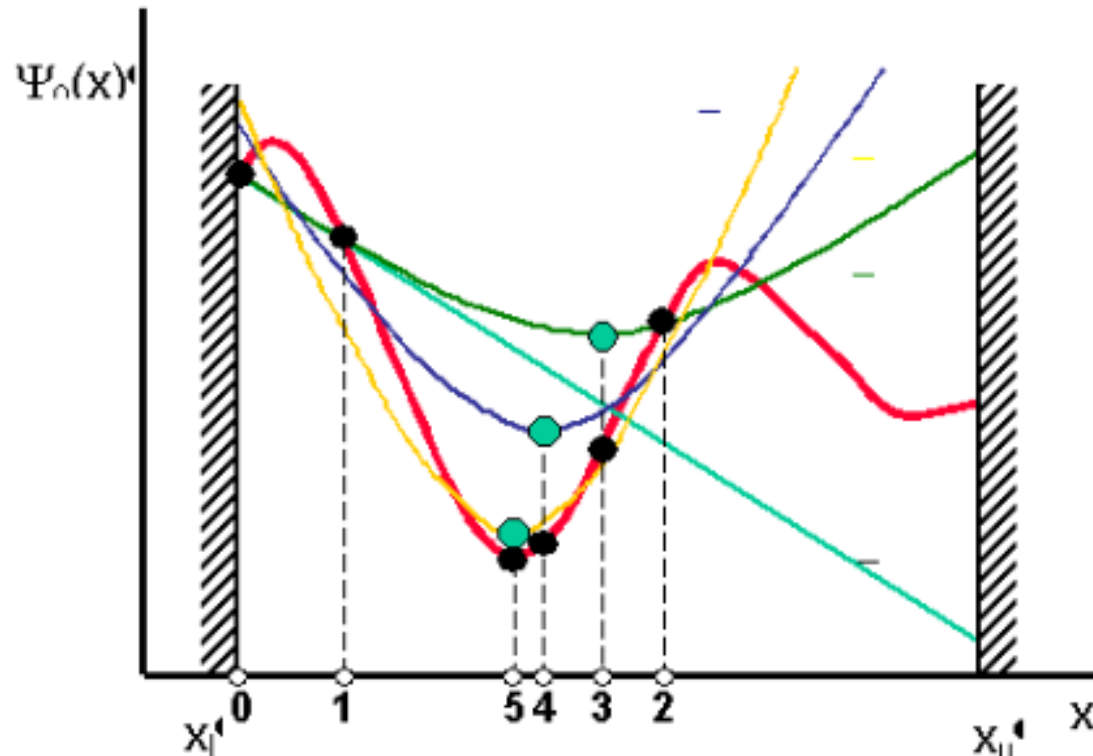
→ 2 constraints on the pressure rise

Define constraints

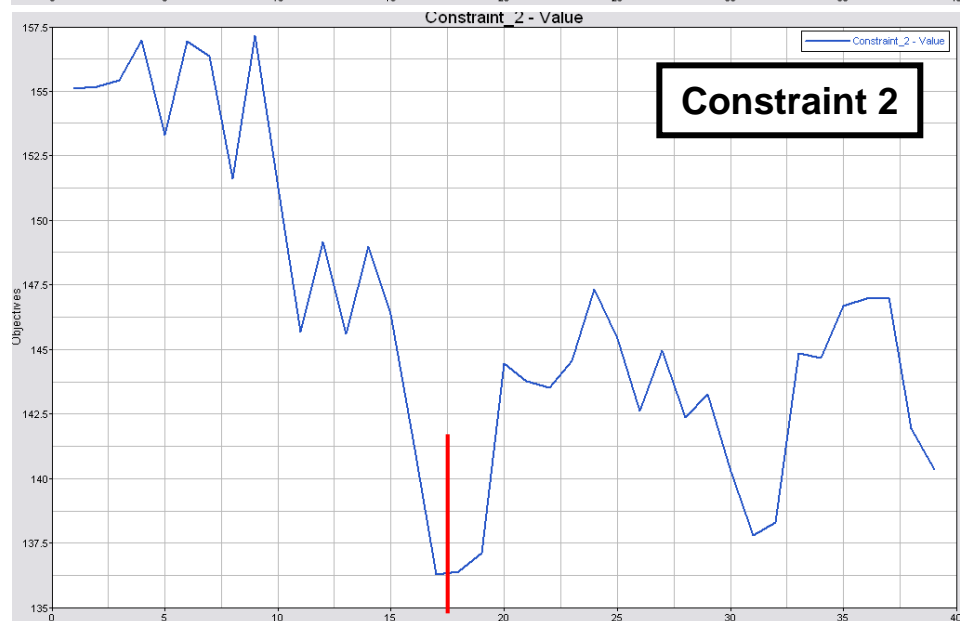
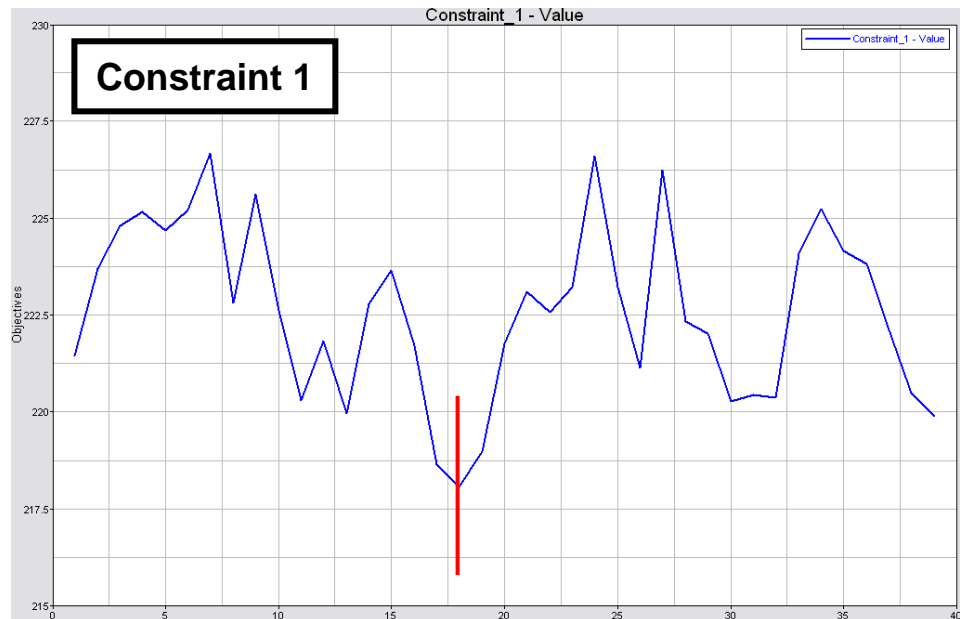
	On	Label	Varname	Type	Apply Constraint On	Bound Type	Bound Value	CDF Limit	Evaluate From
▶	<input checked="" type="checkbox"/>	Constraint_1	c_1	Deterministic	delta_p_rpm2300	>=	210.000000		SOLVER
	<input checked="" type="checkbox"/>	Constraint_2	c_2	Deterministic	delta_p_rpm3500	>=	0.00000000		SOLVER

# Optimisation with Adaptive Surface Response Method (ARSM)

- Objectives and constraints are approximated with a second order polynomial
- The polynomial coefficient are determined using a square fit on previous design points
- Stop when converged below a certain level of error from one iteration to the other



# Convergence of the process



	Pressure rise	Torque	Efficiency (%)
Initial Run	221,45 Pa	1,0416 N.m	54,0
Optimized shape	218,65 Pa	0,9184 N.m	60,5

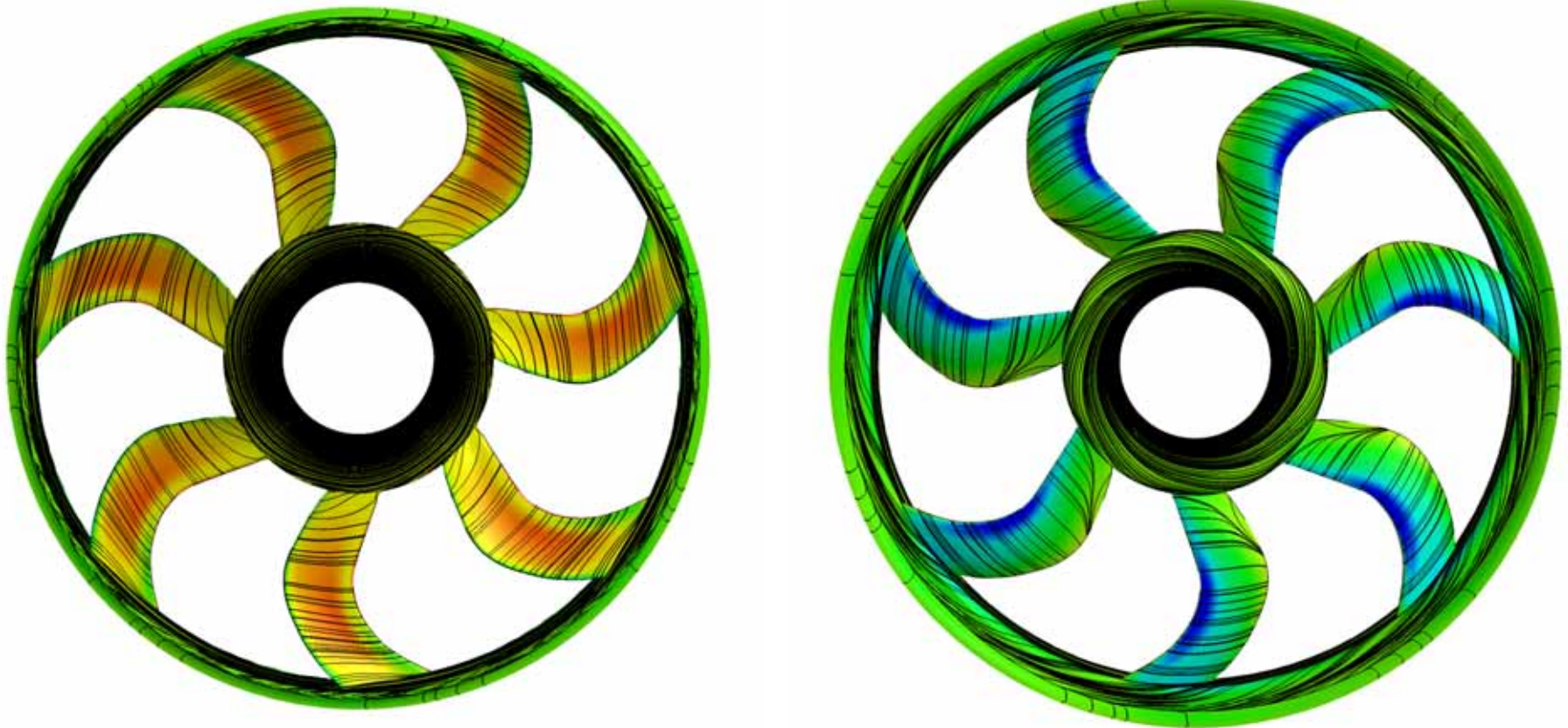
- Great improvement on the objective (+6.5%)
- Constraint are still met
- not so much iterations

## Result: pressure distribution on the optimized blade



- The fan blade is not highly loaded at the top, probably a consequence of the objective of reducing the torque (pressure momentum increases with radius)
- Low load at tip may improve the acoustic

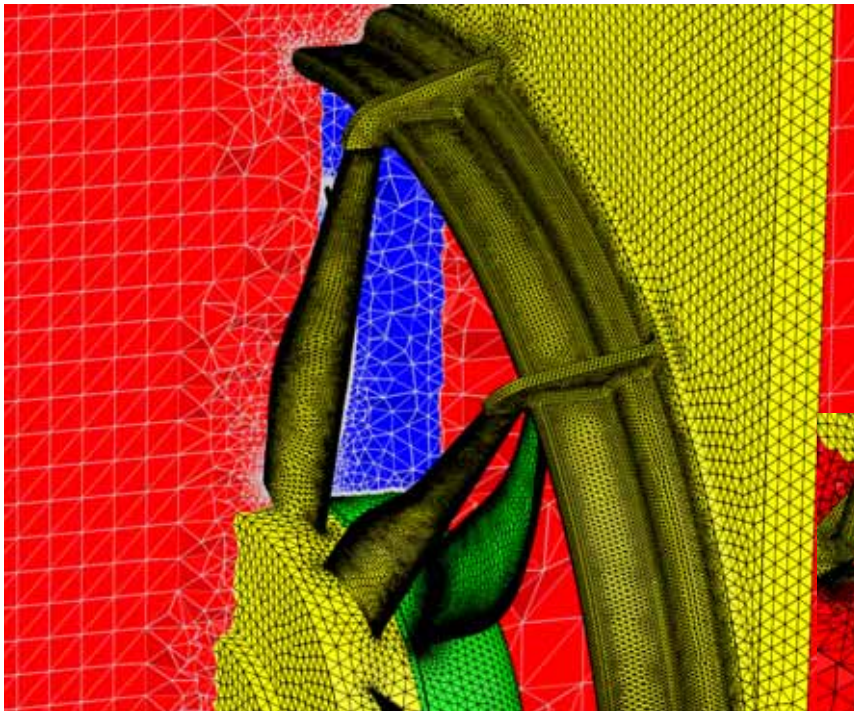
## Result: radial distribution on the optimized blade



- The optimization went naturally to a radially equilibrated fan (concentric flow pattern on the blade), which is in theory the optimum
- Less good at bottom, where parameters were disabled for the reduced set

## 5 – Aeroacoustic

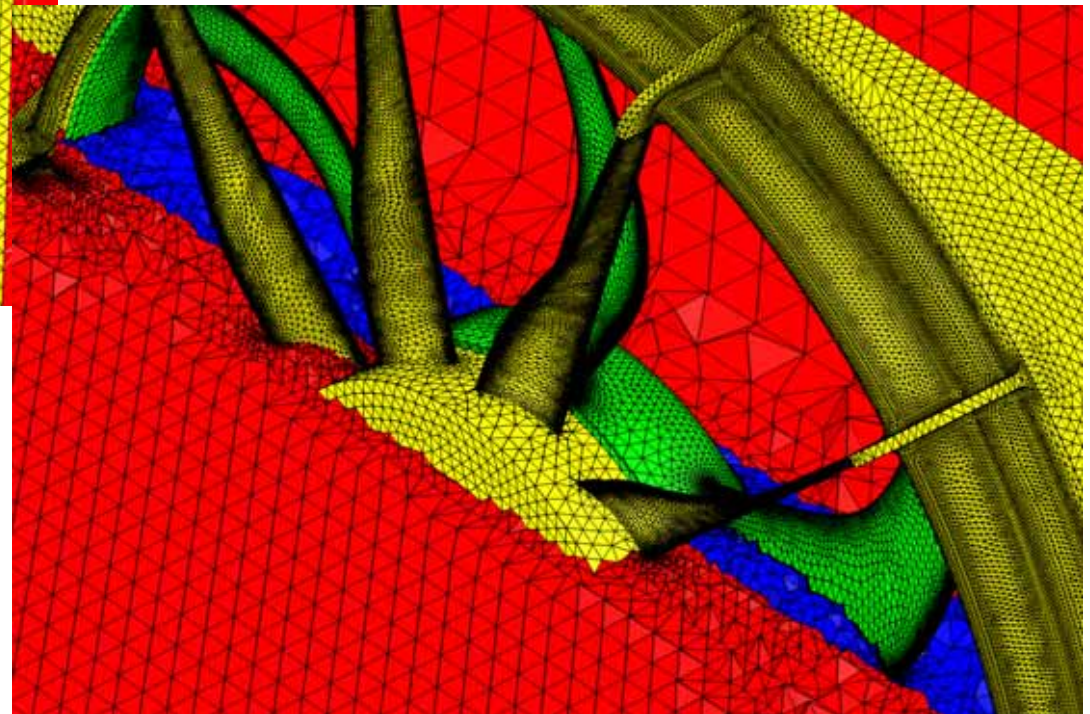
# Aeroacoustic: module simulation (CFD++ / CAA++)



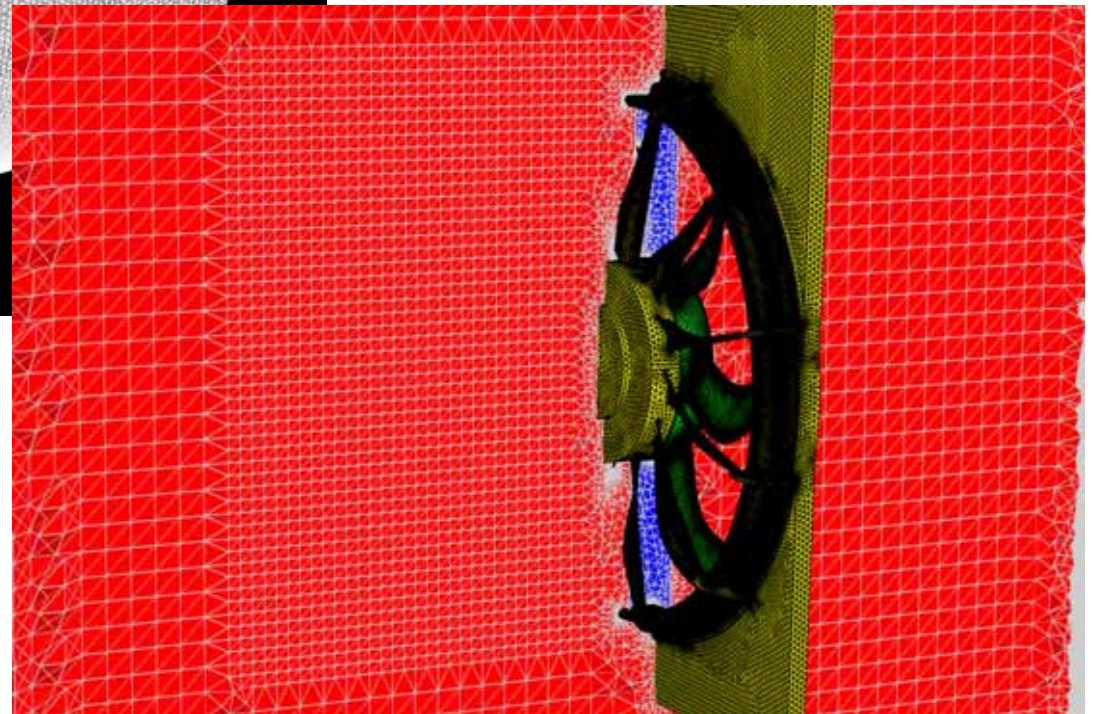
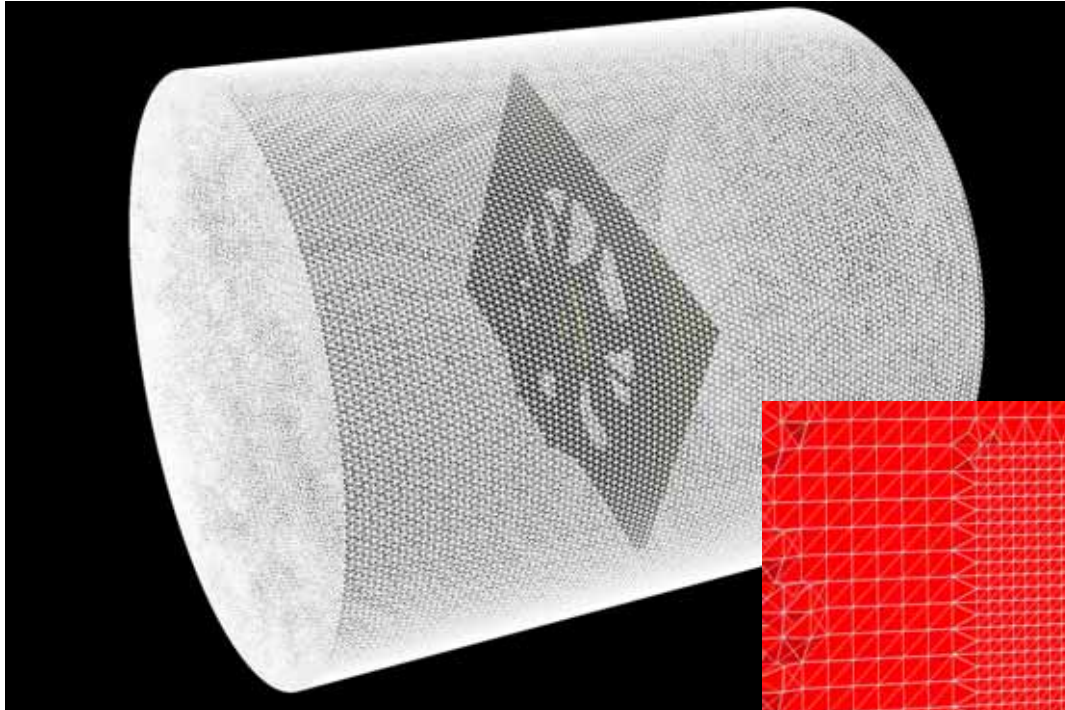
**Aeroacoustic capabilities provided by the simulation:**

→ A complete mesh of the module (rotor-stator interaction) is built to predict pressure fluctuation on surfaces

→ Regular mesh required for propagation



# Aeroacoustic: sound propagation and directivities

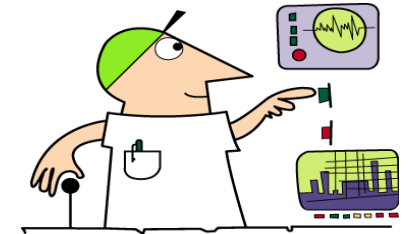


- Tonal noise is predicted from pressure post-processing
- Domain of propagation allows studies on directivities
- Provide data to understand noise mechanism and related geometrical effects

## 6 – Conclusions



# Conclusions: results



## ■ Fan systems optimisation

- Main parameters were chosen by expert in charge of fan system development
- Analysis of sensitivity was conducted to check the relevancy of these parameters
- A reduce set of parameters was used to perform a optimisation

## ■ Optimization results

- A complete loop of optimization has been established on Hyperworks platform linked to CFD++ to evaluate the fan performance
- Despite the initial run started from a blade considered as good, a great improvement (+6,5%) on efficiency was obtained in few iterations

## ■ Gain on the methodology

- The development time can be reduced from one month with the “standard human process” to less than one week with a reduced set of parameters
- High power computing can even improve the process to 2 days of simulation with the complete set of 12 parameters (estimated time, work on progress)

# Conclusions : the role of an optimization process

- **Standard design process is more complicated regarding new specifications**
  - Some risks exist with classical methods: no solution found in the timeframe or below the optimum
- **Optimization process should secure the situation**
  - Establish the methodology helps capture the current know-how and the expertise
  - Several preferred solutions can be evaluated by accurate numerical simulations
- **Need to concentrate our human resources on system analysis**
  - Less human time on fan development
  - Going further in our research (acoustic, module, under hood...)



Product design must be automated by the help of optimization process  
R&D efforts must be focused on innovation



*Enabling a better automotive world*

